



**NECPUC Symposium, Hartford, Connecticut – June 4, 2019**  
New England Conference of Public Utility Commissioners

*Decarbonization through Energy Efficiency*

# ENERGY EFFICIENCY GAMECHANGER



- ***Natural Gas-Powered Heat Pump***
- ***50% Reduction in Energy Use***
- ***Cooling with Natural Gas***
- ***One Appliance – Heating, Hot Water, Air Conditioning and Refrigeration***
- ***Cold Climate Heating with no additional heat source (even below 32 degrees F)***
- ***No Refrigerants & No Toxic Emissions or Materials***

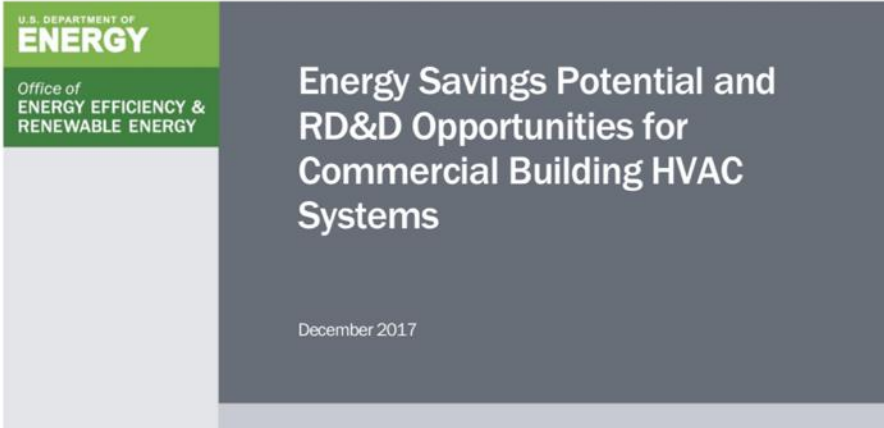




# THERMOLIFT: DOE #1 RANKED HVAC TECHNOLOGY

ThermoLift Heat Pump Highest Ranking

ENERGY SAVINGS POTENTIAL AND RD&D OPPORTUNITIES FOR COMMERCIAL BUILDING HVAC SYSTEMS



Selection based on global review of 300 technologies.

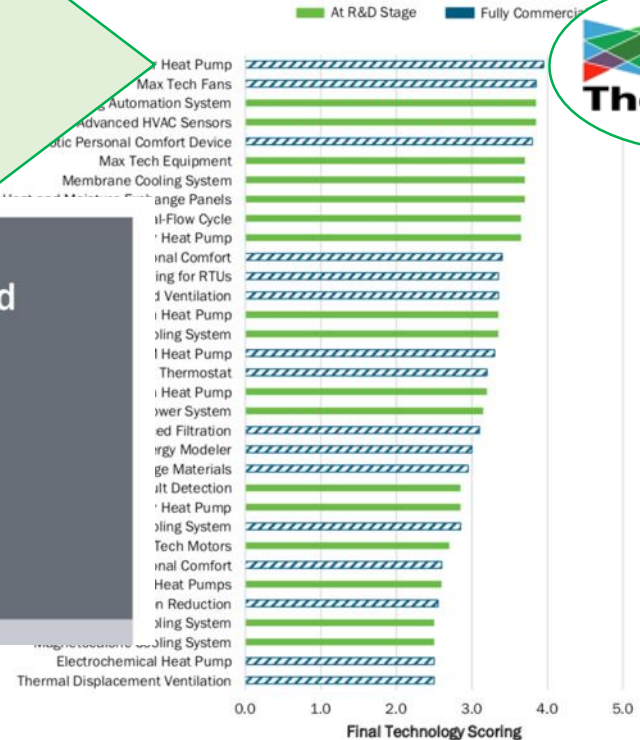
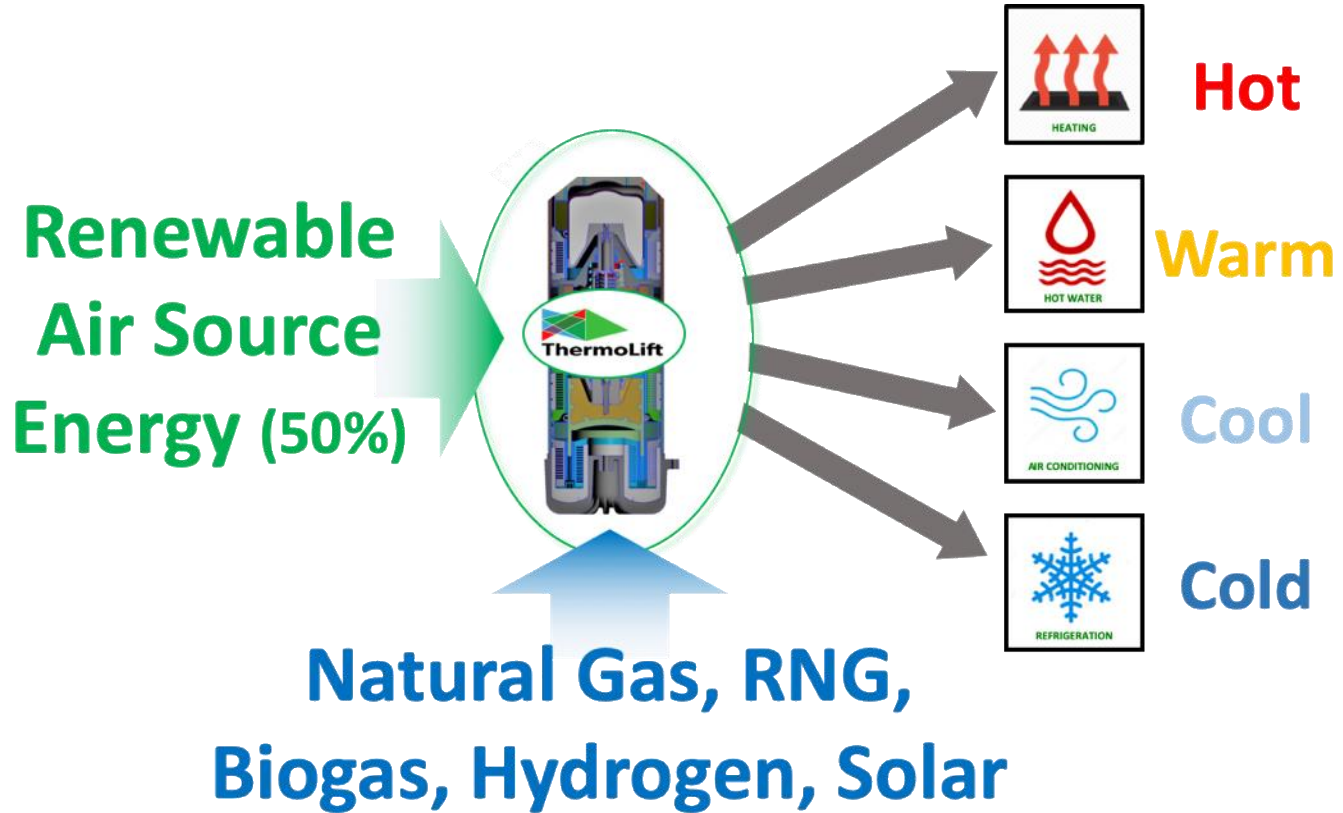


Figure 9: Final ranking of high priority technology options

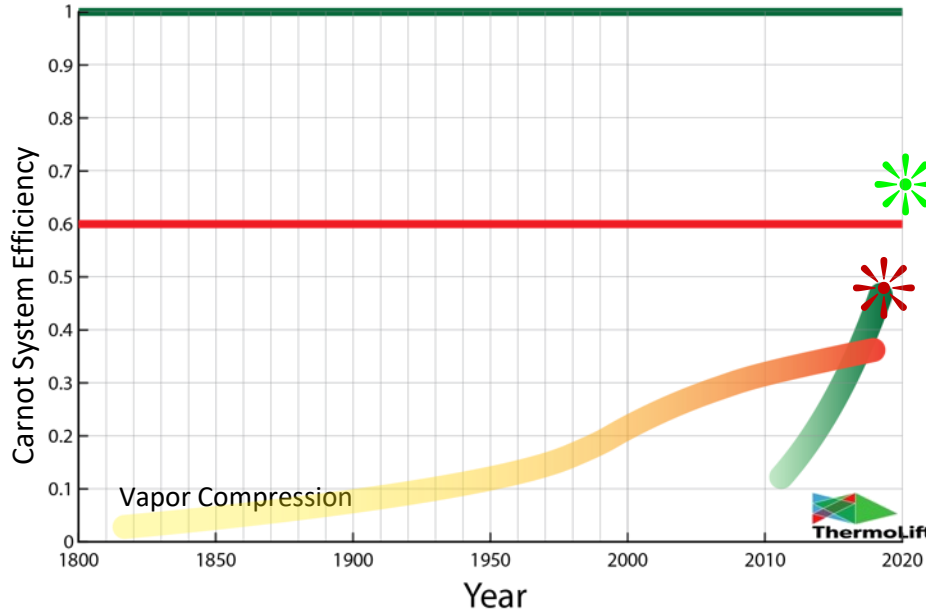
# THERMAL COMPRESSION HEAT PUMP (TCHP™)



# ACCELERATING ENERGY EFFICIENCY INNOVATION

## Superior Efficiency with Zero Refrigerants, Zero HFCs

Already  
Surpassing 200  
Years of Vapor  
Compression  
Technology

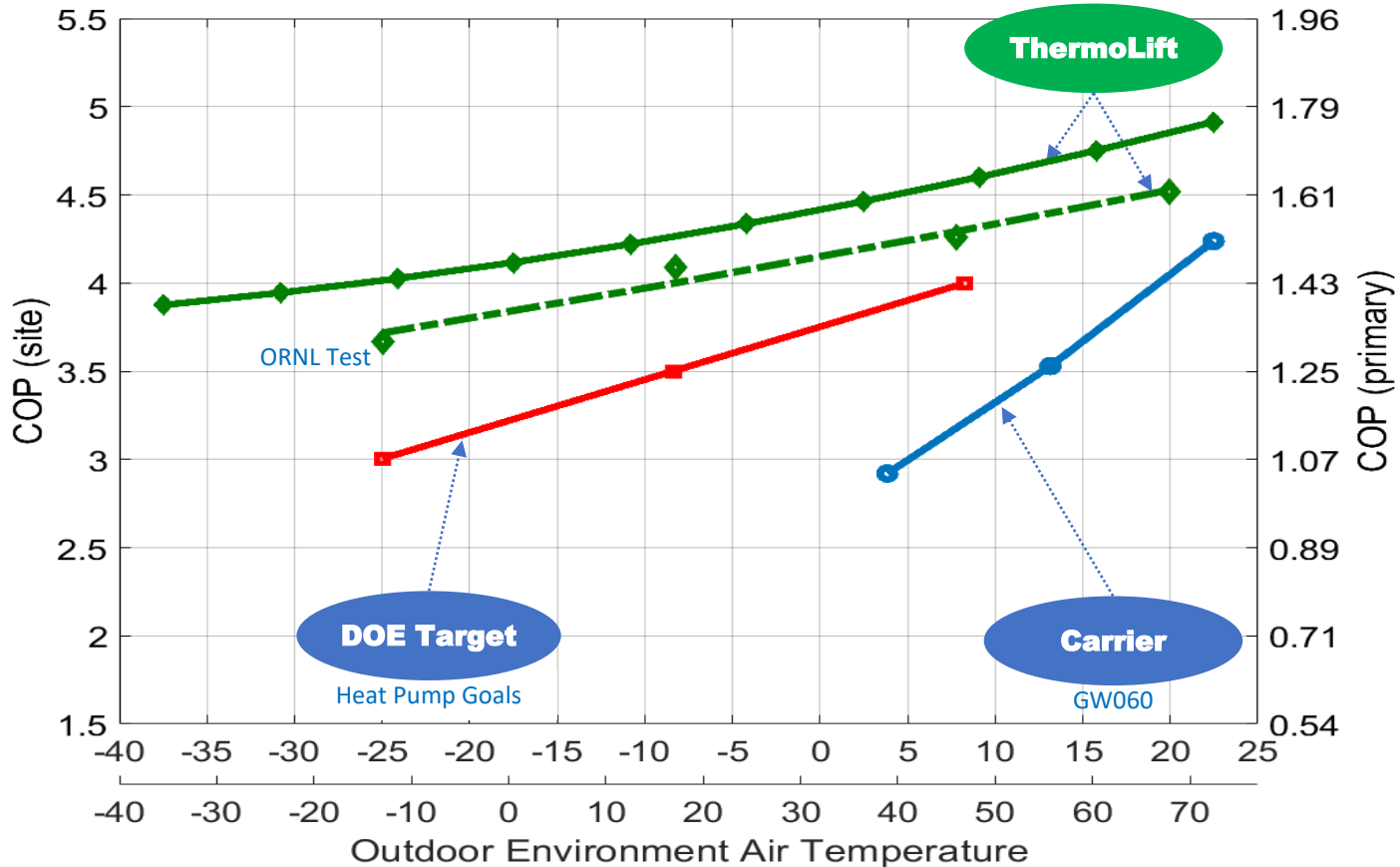


**Soon:** HCTC Exceeds  
Vapor Compression  
Theoretical Potential

**Today:** HCTC Exceeds  
Vapor Compression  
System Level Efficiency

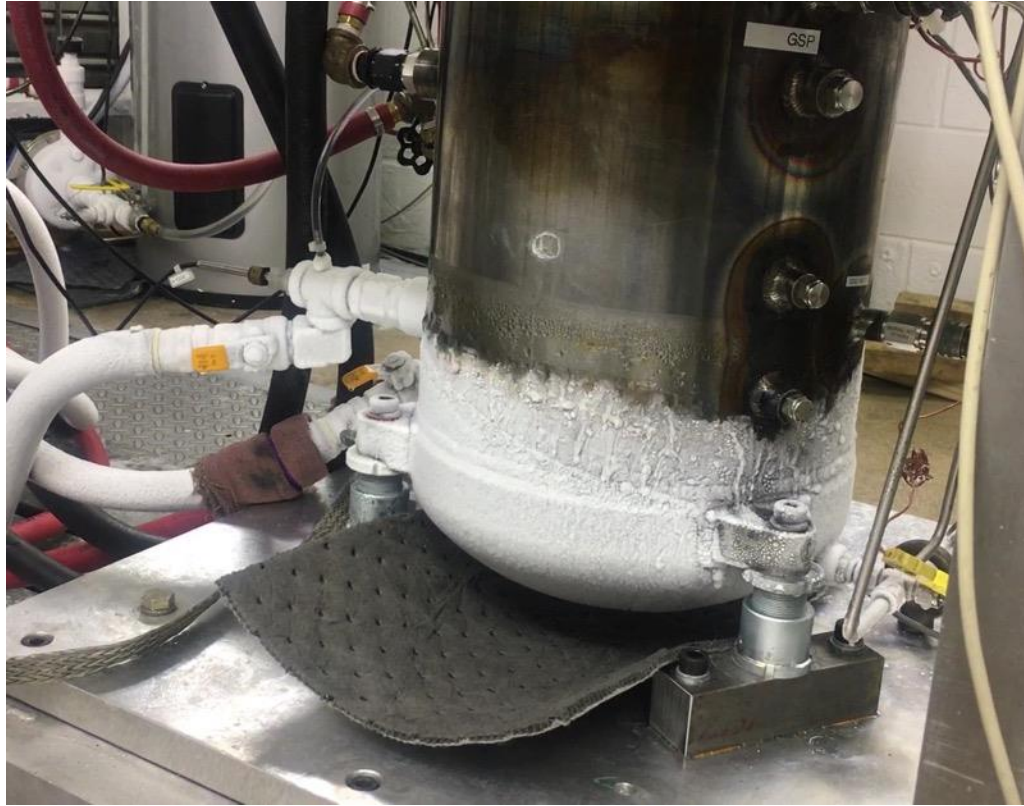
# SIGNIFICANTLY EXCEEDS STATE OF THE ART - IN EFFICIENCY & PERFORMANCE

DOE  
Independent  
Testing at  
Oak Ridge  
National Lab



# LOW TEMPERATURE CAPABILITY – NO BACKUP

*“Polar Vortex”  
Compatible:  
Constant Capacity at  
Cold Temperatures,  
No Backup System  
Needed.*



Monovalent System

-150°F (-100°C)  
demonstrated at  
Oak Ridge  
National Lab



# THERMOLIFT: NO REFRIGERANTS, NO HFCs, SAFE

- ThermoLift **reduces by 100%** the greenhouse gas impact of refrigerants used by current technology (“Vapor Compression”) or approximately 6 tons of CO<sub>2</sub>e over the units lifetime (due to no refrigerants; ThermoLift mitigates another 104 tons of greenhouse gas equivalents due to energy efficiency or 110 tons total).
- ThermoLift’s Thermal Compression (TC-Cycle) uses No Refrigerants, No HFCs.
- Vapor Compression Heat Pumps – globally used for AC, Refrigeration and also Heating, require refrigerants and use HFCs\*.
- HFCs by themselves, left unchecked, are predicted to contribute to a 0.5° C rise in global temperatures by 2100\*\*.

Appliance Type ****	Current Leak Rate	Leak Rate Effective 1/1/2019
Industrial process refrigeration <sup>a</sup>	35%	30%
Commercial refrigeration	35%	20%
Comfort cooling	15%	10%
All other appliances	15%	10%

HFCs, used mainly in refrigeration, air conditioning and heat pump equipment, are thousands of times more harmful to the climate than CO<sub>2</sub>. In response to the rapid growth of HFC emissions, the 197 parties to the Montreal Protocol adopted the Kigali Amendment in 2016 to reduce gradually their global production and consumption.\*\*

## KEY FIGURES \*\*\*

1,430x	10-15%	29 years	5.6 billion
The most abundant HFC is 1,430 times more damaging to the climate than carbon dioxide per unit of mass	Emissions of HFCs are growing at a rate of 10-15% per year	HFCs remain in the atmosphere for up to 29 years	The global stock of air conditioners in buildings will grow to 5.6 billion by 2050, which amounts to 10 new units sold every second for the next 30 years

## ThermoLift Safety, Installation, Operations Benefits:

- Simplified operation & maintenance due to no refrigerants.
- Infinite zoning capability.
- No restrictions on cooling line lengths.
- No dangerous alternatives with indoor building code restrictions, such as flammables, ammonia, or high pressure CO<sub>2</sub>.
- ThermoLift resolves ASHRAE 15 & 34 compliance installation issues for safety.
- No EPA 608 installer certification needed.

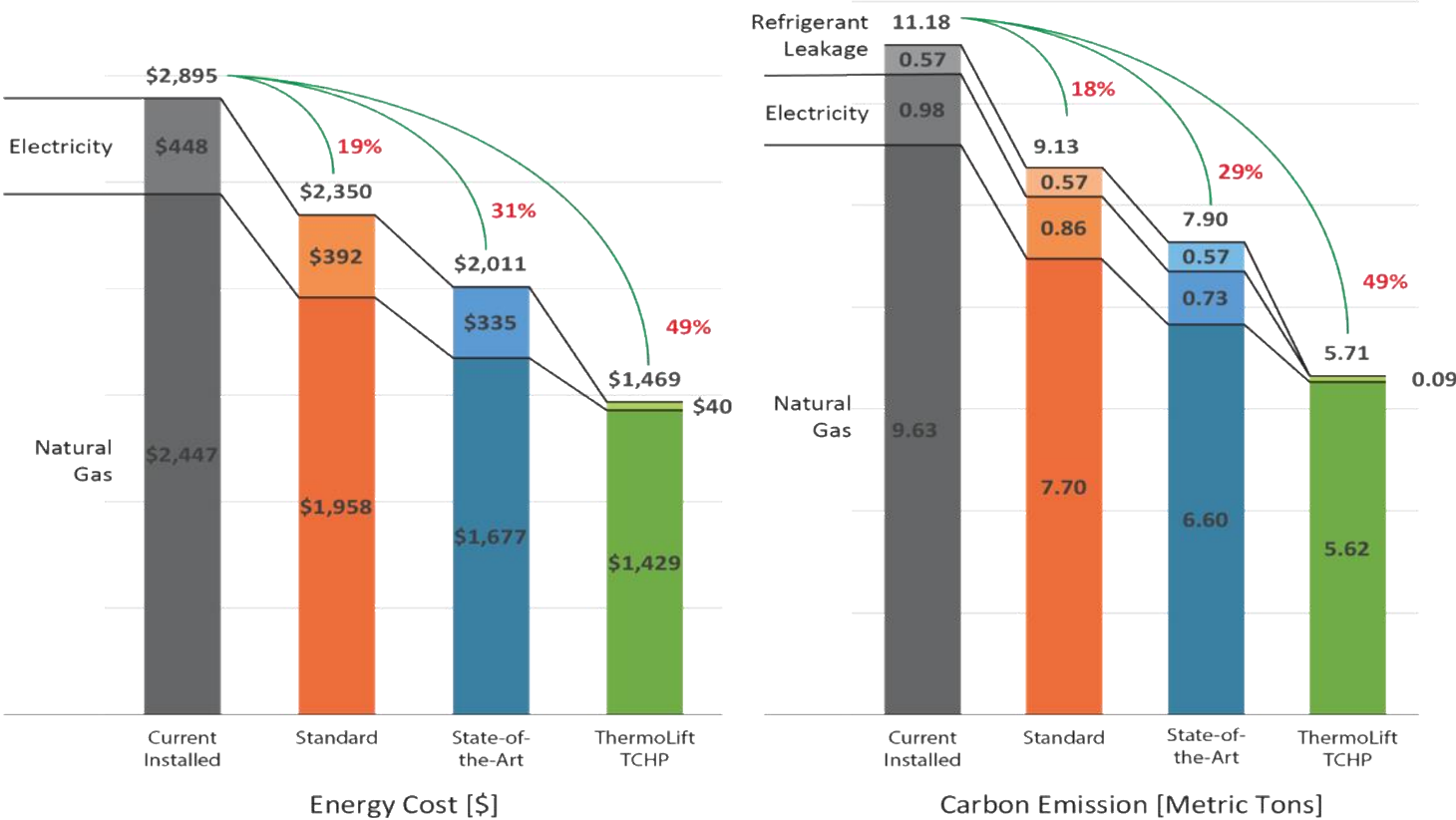
\* HFCs – Hydrofluorocarbons, are the most common type of refrigerant chemicals.

\*\* Kigali Agreement: [https://ec.europa.eu/clima/news/eu-ratifies-kigali-amendment-montreal-protocol\\_en](https://ec.europa.eu/clima/news/eu-ratifies-kigali-amendment-montreal-protocol_en)

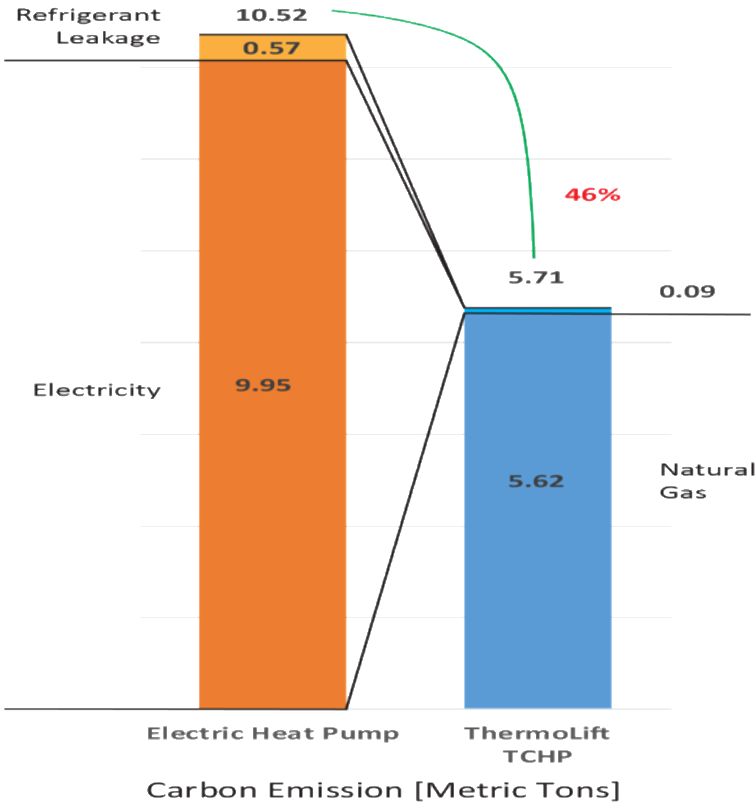
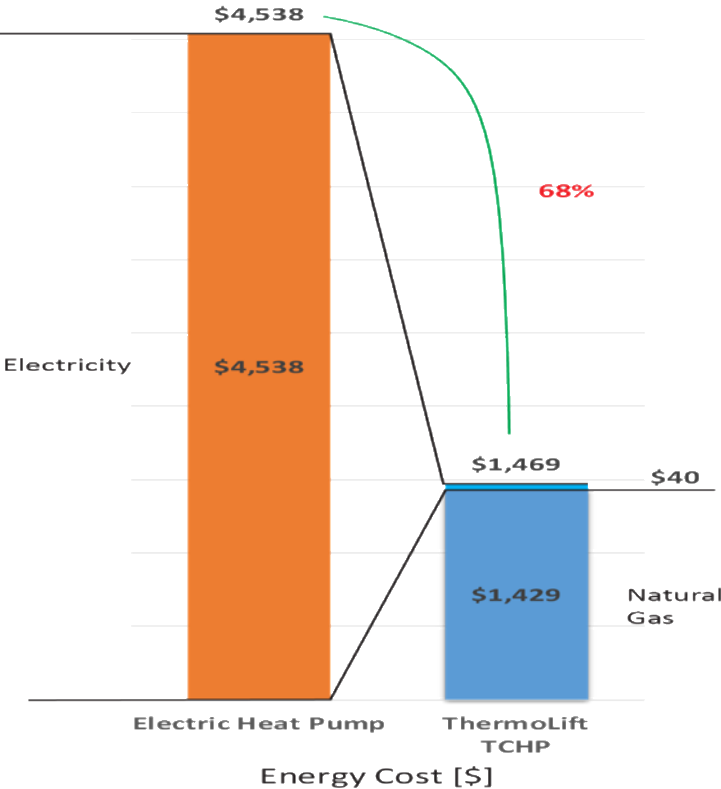
\*\*\* <http://ccacoalition.org/en/sicps/hydrofluorocarbons-hfc>

\*\*\*\* EPA allowed refrigerant leakage rates: <https://www.epa.gov/section608/stationary-refrigeration-leak-repair-requirements>

# Hartford, CT – Residential Energy & CO2 Reduction



# Hartford, CT – Residential Heat Pump Comparison



# ThermoLift Competitive Landscape

	ThermoLift TCHP™	Condensing Boiler	High Efficiency AC (13 SEER)	Air Source Heat Pump (Minisplit VRF)	Water Source Heat Pump	Geothermal Source Heat Pump	Absorption Heat Pump
<i>Heating</i>	●	●	●	●	●	●	●
<i>Cooling</i>	●	●	●	●	●	●	●
<i>No Refrigerants</i>	●	●	●	●	●	●	●
<i>Renewable Energy</i>	●	●	●	●	●	●	●
<i>Hot Water</i>	●	●	●	●	●	●	●
<i>Efficient Heating below 0°C (32°F) Ambient Outdoor</i>	●	●	●	●	●	●	●

# ThermoLift System Benefits



## Consumer Benefits:

- ✓ *Lower Operating Cost*
- ✓ *High Performance*
- ✓ *Standard Installation Cost*
- ✓ *Renewable Energy*
- ✓ *Compact / Retrofit*
- ✓ *Support Fuel Switching from Oil*

## Non-Pipe Solution for Capacity Constraints:

- ✓ *Expand Capacity through 50% End Use Efficiency*
- ✓ *No New Infrastructure Cost*
- ✓ *Address Seasonal Constraints and Moratoriums*

## Cold Climate High Efficiency:

- ✓ *Constant Capacity even at Very Cold Outside Temperature*
- ✓ *No Need for Backup Heating*
- ✓ *Meet Peak Demand Heating without Additional Capacity - "Polar Vortex" Compliant*

## Demand Response:

- ✓ *Gas Demand Response*
- ✓ *Variable Performance Range*
- ✓ *High Efficiency Modulation without Cycling On/Off*

## No Refrigerants:

- ✓ *1,500x – 2,000x Worse Greenhouse Gas Impact*
- ✓ *Being Phased Out*
- ✓ *Leak Rates*

## Decarbonizing through Energy Efficiency:

- ✓ *Lower Carbon Footprint*
- ✓ *Cost Effective GHG Reduction*
- ✓ *Important Solution towards Achieving Mandates, Regional GHG & Climate Impact Targets*

# PARTNERS, COLLABORATION, SUPPORT

## DEVELOPMENT AND DEMONSTRATION



IN<sup>2</sup> Program



## NATIONAL LABORATORIES



## ACADEMIC PARTNERS



## GLOBAL MANUFACTURING POWERHOUSE



**\$7B Tier 1 Global Manufacturer**

- 60 Plants Worldwide
- Reliability & Warranty
- Cost & Scale

## DISTRIBUTION, INSTALLATION AND SERVICE / SUPPORT RELATIONSHIPS



**To Be Announced**



**To Be Announced**

# STRONG IP PORTFOLIO & RECOGNITION

## RECOGNITION

- DOE #1 Ranked HVAC Technology
- NREL IN<sup>2</sup> Incubator
- IEA Heat Pump Publication
- DOE Report on HVAC Technologies
- 2015 Long Island's Innovator of the Year
- American Gas Magazine
- NREL Industry Growth Forum
- Stony Brook University Start-Up of the Year
- CEBIP Incubator Company of the Year

## GRANTS

- Department of Energy
- NYSERDA
- Gas Technology Institute
- Wells Fargo IN<sup>2</sup>
- NYS Manufacturing

## STRONG INTELLECTUAL PROPERTY PATENT PORTFOLIO

17 patent families; 15 patents issued; 31 patents pending. Trademark protection has been filed and allowed.

PATENT FAMILIES	DATE	STATUS
Heat Pump with Electromechanical Actuated Displacers	4/11/2013	<b>Issued:</b> US, China, Canada <b>Filed:</b> Europe, India
Combination Solar and Combustion Heater	10/18/2013	<b>Issued:</b> Canada <b>Allowed:</b> China <b>Filed:</b> Europe, India, US
A Compact Heat Exchanger for a Heat Pump	11/25/2013	<b>Filed:</b> China, Germany, UK, US
A Combination Heat Exchanger and Burner	12/4/2013	<b>Filed:</b> China, Canada, Europe, India, US
A Four-Process Cycle for a Vuilleumier Heat Pump	11/18/2014	<b>Allowed:</b> China <b>Filed:</b> Canada, Europe, Korea, US
A Vuilleumier Heat Pump Having a HX Located Between the Displacers	2/21/2015	<b>Filed:</b> China, Denmark, US
A Heat Exchanger	6/10/2016	<b>Filed:</b> US
A Spring for an Electromagnetic Actuator System	9/15/2016	National phase filing 3/15/2018
Dome for a Thermodynamic Apparatus	10/15/2016	National phase filing 4/15/2018
Gas Spring and Bridge for a Heat Pump	10/18/2016	National phase filing 5/18/2018
Mechatronic Drivers in the Cold End of a Heat Pump	10/18/2016	National phase filing 5/18/2018
A Regenerator	3/16/2016	<b>Filed:</b> US
Spiral Extruded Heat Exchanger	2/22/2017	National phase filing 8/22/2018
A Linear Actuation System Having Face Coils and Side Coils for Armature Travel Assist	4/8/2017	File PCT or national phase by 4/8/2018
A Linear Actuation System Having Side Stators and a Permanent Magnet Armature	4/24/2017	File PCT or national phase by 4/24/2018
Centrally-Located Linear Motors for Driving Displacers in a Thermodynamic Apparatus Regenerator	9/25/2017	File PCT or national phase by 9/25/2018

# WORLD CLASS LEADERSHIP



**Paul Schwartz**  
**CEO, Co-Founder**

20+ years of  
experience in finance,  
investment banking.



**Prof. Dr.-Ing.  
Peter Hofbauer**  
**President,  
Co-Founder**

Former VP Viessman,  
European  
Environmental Award,  
Former Global Head of  
Engine & Powertrain  
VW



**Robert Catell**  
**Board of  
Directors**

Former Chairman of  
National Grid (US)  
and CEO of KeySpan  
  
Chairman of the  
American Gas  
Association and US  
Energy Association



**Steve Winick**  
**Board of  
Directors**

Partner at Topspin  
Partners  
  
Former CTO of  
Honeywell's \$2B  
Home Security  
Group



**David Parks, PhD**  
**Board of  
Directors**

HVAC Industry  
Leader  
  
Former C-Level  
Positions in HVAC  
including Carrier,  
Haier America and  
Goodman

*25 Full Time Team Members, including 9 PhDs, plus Extensive & Active Senior Advisor Network*



# THERMOLIFT: Energy Efficiency Gamechanger

## Advanced Innovation to Meet Real Needs of NE Region:

- *Gas-Powered Air Source Heat Pump for Heating & Cooling*
- *DOE #1 Ranked HVAC Technology*
- *Up to 50% Reduction in Energy Use & Greenhouse Gas Emissions*
- *Deliver Significant Consumer & System Benefits*
- *No Refrigerants*

## Seeking:

- *Rebate Program Participation*
- *Installation Incentives & Support*
- *Partnerships, Collaborations, Investors*



# THANK YOU !

QUESTIONS ?



# ThermoLift

*THE ULTIMATE HEAT PUMP™*

**Paul Schwartz, CEO**

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S U P P L E M E N T A L   S L I D E S



ThermoLift at OGCI Climate Initiative, Houston, 2019

# Peer Reviewed Principles of Operation

Detailed thermodynamic engineering analysis in Applied Thermal Engineering Journal

H. Chen et al. Applied Thermal Engineering 140 (2018) 553–563

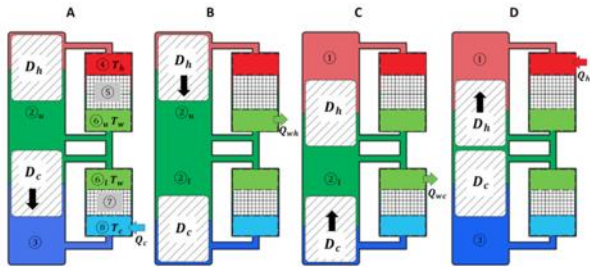


Fig. 3. VM cycle principle of operation.

Table 2 Assumptions for the model.

Component	Temperature in component (constant)	Gas entering	Gas leaving	Heat flow in one cycle
Hot HX $\textcircled{4}$	$T_h$	$T_1 \textcircled{1} \rightarrow 0$ $T_2 \textcircled{2} \rightarrow 0$	$T_w \textcircled{3} \rightarrow 0$ $T_c \textcircled{4} \rightarrow 0$	$Q_h$ (in)
Warm HX $\textcircled{5}$	$T_w$	$T_1 \textcircled{1} \rightarrow 0$ $T_w \textcircled{2} \rightarrow 0$	$T_w \textcircled{3} \rightarrow 0$ $T_w \textcircled{4} \rightarrow 0$	$Q_w$ (out)
Cold HX $\textcircled{6}$	$T_c$	$T_1 \textcircled{1} \rightarrow 0$ $T_c \textcircled{2} \rightarrow 0$	$T_c \textcircled{3} \rightarrow 0$ $T_c \textcircled{4} \rightarrow 0$	$Q_c$ (in)
Hot Regenerator $\textcircled{7}$	$T_h \pm \Delta T_r$	$T_1 \textcircled{1} \rightarrow 0$ $T_w \textcircled{2} \rightarrow 0$	$T_w \textcircled{3} \rightarrow 0$ $T_h \textcircled{4} \rightarrow 0$	0
Cold Regenerator $\textcircled{8}$	$T_c \pm \Delta T_r$	$T_1 \textcircled{1} \rightarrow 0$ $T_c \textcircled{2} \rightarrow 0$	$T_c \textcircled{3} \rightarrow 0$ $T_w \textcircled{4} \rightarrow 0$	0

$$COP = \frac{T_w(T_h - T_c)}{T_h(T_h - T_c)} \quad (1)$$

## 2.2. Non-ideal adiabatic model (NAM) assumptions and approach

The model assumptions are stated as follows:

- The gas chambers  $\textcircled{1}$ ,  $\textcircled{2}$ ,  $\textcircled{3}$  are adiabatic. Their temperatures,  $T_1$ ,  $T_2$  and  $T_3$  will vary as the cycle proceeds.
- The HXs  $\textcircled{4}$ ,  $\textcircled{5}$  and  $\textcircled{6}$  are assumed to be ideal, isothermal heat reservoirs. They have a finite volume and permit heat flow.
- The regenerators  $\textcircled{7}$ ,  $\textcircled{8}$  are also assumed to have a constant average temperature based on the average of the inlet and exit temperatures.
- All mechanical motion is frictionless.
- There is no internal heat leakage by conduction in the machine.
- The helium behaves as an ideal gas.

Note that the temperature of the gas leaving the HXs and regenerators depends on the direction of the flow. Table 2 summarizes the heat flows and the inlet/exist temperatures for the HX regenerators, depending on the direction of the flow.

Applied Thermal Engineering 140 (2018) 553–563



Contents lists available at ScienceDirect

Applied Thermal Engineering

journal homepage: www.elsevier.com/locate/apthermeng



Research Paper

## Performance analysis of a free-piston Vuilleumier heat pump with dwell-based motion

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<sup>a</sup> Department of Mechanical Engineering, Stony Brook University, NY 11790, USA

<sup>b</sup> ThermoLift, Inc., Stony Brook, NY 11790, USA



H. Chen et al.

Applied Thermal Engineering 140 (2018) 553–563

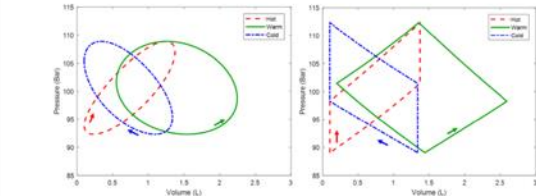


Fig. 8. PV diagram of S-motion and D-motion.

Table 4 Simulation results with hot, warm and cold temperatures of 700, 70 and 0 °C, respectively.

No.	Case	Hot input (kW)	Warm output (kW)	Cold input (kW)	Heating COP
1	S-motion	4.93	11.64	0.72	2.36
2	D-motion	4.44	14.86	0.42	2.31

$$\Delta P = L_f \frac{\rho v^2}{2d} \quad (25)$$

Where  $\rho$  is density,  $v$  is the average gas velocity,  $L_f$  is the regenerator length,  $f_f$  is the friction factor and  $d_f$  is the hydraulic diameter of the regenerator.

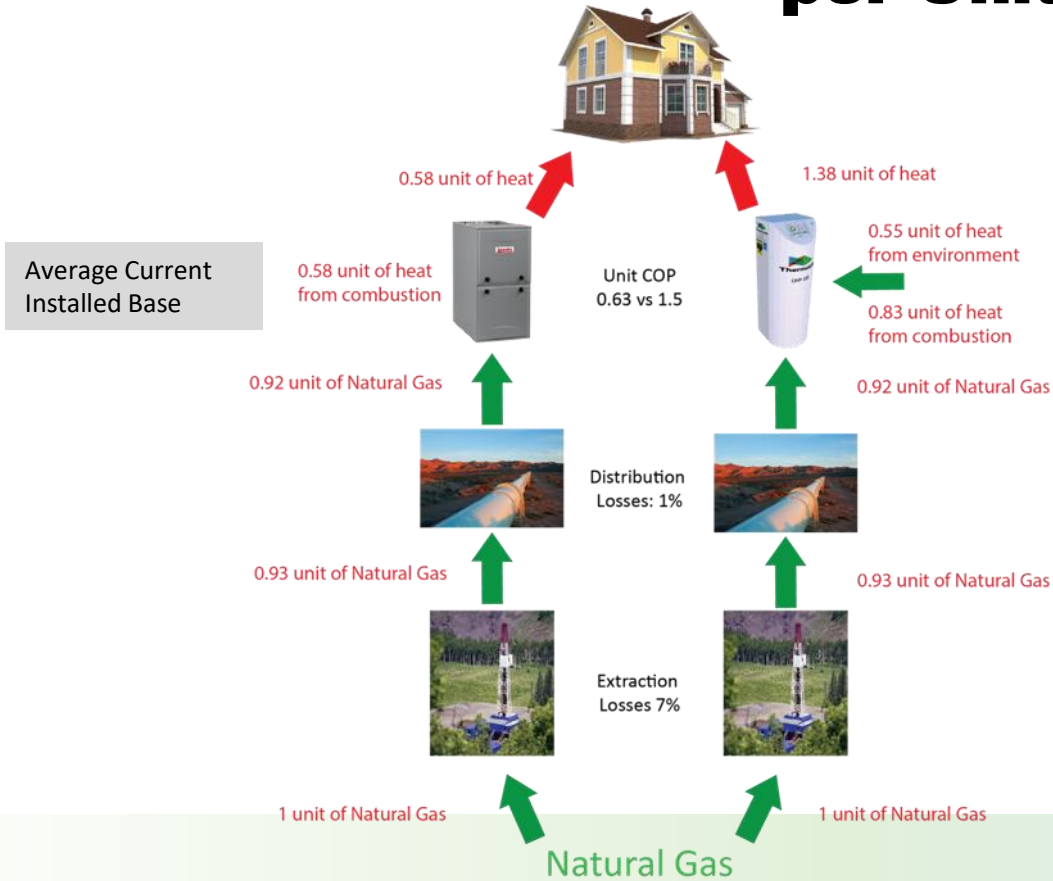
4. Driving-rod PdV work. A cylindrical rod is attached to each displacer for motion control, and to provide alignment in the cylinder bore. The rods produce a small amount of PdV work as the dis-

### 3.1. Comparison of D-motion and S-motion

The hot and cold displacer positions are shown for both S-motion and D-motion over one complete cycle in Fig. 5. By convention, the results for one complete PVHD cycle are expressed in terms of an equivalent crankshaft angle ranging from 0° to 360° (4,19). The amplitude of all motions is  $\pm 25$  mm. The model was run using the TL-PVHD specifications from Table 3 for both cases.

The D-motion is realized by a closed-loop control system consisted of springs and electromagnets. Proper spring configuration ensures that the displacer will move to  $\pm 25$  mm of the chamber at the end of each phase with a minimal energy input. Once the displacer comes to rest, an electromagnet holds the displacer in position during its dwell cycle. Only a small amount of electricity is required, since the distance between the displacer and the electromagnet is small. The energy required to overcome friction and viscous effects in the helium comes from the small amount of rod PdV work discussed above. The machine can also be fitted with a small magnet-coil assembly on one of the displacers to produce the required electrical energy to operate the machine. This

# ThermoLift Delivers 2.4x More Heat per Unit of Natural Gas



ThermoLift delivers more value for energy.



Units: all on equivalent MMBtu basis

COP: Coefficient of Performance = Heat Delivered / Fuel Input



# ThermoLift Renewable Energy ‘Capacity Factor’\*

	Wind	Solar	ThermoLift Heat Pump	Electric Heat Pumps
Capacity Factor	37%**	26%**	100%***	~ 50%***
Renewable Energy	100%	100%	50% - 100%	50% - 100%
Refrigerants or HFCs	No	No	No	Yes, Refrigerants @ up to 2,000x CO2 GHG Impact

\* Though Heat Pumps are renewable energy devices, calculating their Capacity Factor (as with wind and solar) is not yet standardized. This analysis considers Heat Pump Capacity Factor (for Heating) as the % of time that the device delivers Renewable Energy for Heating (i.e. does not rely on a secondary heat source, or does not utilize renewable energy for heating) during heating days, following the similar logic for wind and solar.

\*\*US Energy Information Administration (EIA), 2018

\*\*\* Calculated from National Oceanic and Atmospheric Administration (NOAA) Data for 2018, Albany Airport Weather Station Temperature Data

# ThermoLift Comparative Analyses

- The building energy simulations are performed using EnergyPlus. EnergyPlus<sup>1</sup> is DOE's flagship whole building energy modeling engine. It is certified by *ASHRAE 140: Standard Method of Test for the Evaluating of Building Energy Analysis Computer Programs*.
- The residential case refers to a typical standalone residential house complying with IECC 2006 code. This building model is developed by DOE under Building Energy Code Program<sup>2</sup>.
- For carbon reduction, the direct carbon emission from natural gas end use and indirect carbon emission from electricity consumption are calculated based on *Greenhouse Gas Emissions*<sup>3</sup> from Energy Star Portfolio Manager. Due to the weather related nature of air conditioner, its consumption is regarded as non-baseload and calculated accordingly.
- Other data based on Energy Information Agency (EIA) and similar databases. Further details available upon request.

1. EnergyPlus: <https://www.energy.gov/eere/buildings/downloads/energyplus-0>

2. DOE Building Energy Codes Program: <https://www.energycodes.gov/development>

3. Energy Star Portfolio Manager: <https://www.energystar.gov/buildings/tools-and-resources/portfolio-manager-technical-reference-greenhouse-gas-emissions>